Anti-Roll Mechanism for Commercial Vehicle

Mahmmadabararkhan N. Diwan¹, Dhruv R. Patel², Dhairya U. Patel³, Divyesh M. Rane⁴, Kritik M. Patel⁵

^{1,2,3}Student, Department of Automobile Engineering, Knowledge Institute of Technology and Engineering, Bakrol, Anand, Gujarat (388315).

^{4,5}Student, Department of Mechanical Engineering, Knowledge Institute of Technology and Engineering, Bakrol, Anand, Gujarat (388315). ***-------

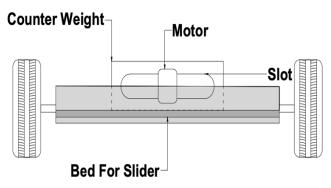
Abstract - Commercial vehicles are inevitable part of the society nowadays as the population is increasing drastically. Commercial vehicle are not only transporting Goods but also the humans. But the accidents in commercial vehicles are increased drastically. There are plethora of causes for accidents but one of which is vehicle rollover. Due to heavy parts and high load carrying there is always a tendency of vehicle to rollover at slopy road or at high speed. According to the report almost around 2.5 lacks trucks got roll overed in 2018 only in India. So, by putting an external counter weight which continuously changes its position according to weight transfer these accidents can be prevented.

Hence, the safety in the automotive domain can be improved by balancing vehicle internally so it can prevent the external arbitrary forces. Thus this research paper focuses on the anti-roll balancing mechanism for the commercial vehicles or heavy-duty vehicles having more play load and goods load.

Key words: Sliding mechanism, heavy-duty vehicle, Antiroll, Counter weight, Position sensors, Chain transmission

1. INTRODUCTION

The rollover of the commercial vehicle (Trucks, Buses, dumper etc.) due to the heavy weight is one of the most common accident causes. The vehicle tends to roll over at sharp corner, at high speed or at very high side slope of the road as the centripetal force is acting on the C.G. continuously. Unlike the other accidents in this there are more chances of total breakdown of the vehicle and the injury of driver and passenger as well. This problem can be prevented by attaching the counter weight mechanism which slides continuously beneath the deck in opposite direction of the centrifugal force. The counter weight mechanism not only counters the centrifugal force but also lower downs the C.G. which aids in stability against the unbalanced forces. The mechanism takes the input from the position sensors and runs the motor in the appropriate direction causing the pulling of the chain attached to the counter weight hence due to the movement of the weight vehicle can be balanced. There are multiple numbers of the motors which pulls the counter weight and the specifications are depends upon the weight of the vehicle and counter weight. The displacement of the slab also depends upon the weight of the vehicle and also the required C.G. transfer in order to balance the vehicle properly. For example if the side slope of the road is 5° then the displacement required is less than it requires for the 10°, and the load plays the major role regard with the angle of slope.





As shown in Fig.1 the block or the counter weight has the slots(geared slots) in which the motors are fitted so when the motor rotes that causes the slab to move in the longitudinal direction. In other word this slots will help the mechanism to translate the rotary motion of the motor to the longitudinal motion of the block. There are few elements which plays major role besides the counter wight and motor which are sensors, Microcontroller and motor controller.

2. COMPONENTS

1.1 Steel Block

It is one of the main components of the system as it counters the forces which tend to roll-over the vehicle. Steel block is used because of its moderate density hence the block will not be too little so the weight concertation causes unwanted forces or not too big to occupy more space in the vehicle.



Fig - 2 Steel block

We know that the density of the steel is between $7355-8050 \text{ kg/m}^3$.

By taking 8000 kg/m³ the dimensions of the block for 240 kg (\cong 243.76 kg) are 500mm*250mm*250mm.

(However the machining is required on this block in order to have groves for the motor to slide it.)

1.2 Motor

Electric motor is used to move the block in lateral directions to the center. Two motors are used in this case as its relatively light commercial vehicle and the weight required to balance it is less. For heavier vehicle number of motors can be used however.



Fig - 3 BLDC motor

Table 1

Specifications:

Voltage	60V
Rated Current	45A
Speed	3000±100
Max. torque	22Nm
Max. Power output	3000 W
Number of poles	8
Efficiency	>83%

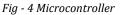
1.3 Microcontroller

Arduino Mega 2560 Rev3(A000067)

Microcontroller takes input from the gyro sensor about the position of the vehicle deck and then it will

send signals to the motor controller about the direction of rotation and speed of the motor.





Specifications:

Table 2

Operating Voltage	5V
Digital I/O pins	54
Analog input pins	16
SRAM	8 KB
EEPROM	4 KB
Clock Speed	16 MHz

1.4 IMU

An Inertial Measurement Unit is an electronic device that measures and reports a body's angular rate, specific force and orientation of body using combination of gyroscope and accelerometer.

NAME: REES52 GY-521 Mpu6050 Module



Fig - 5 IMU

1.5 Motor controller

Motor controller is the device used to control motor by running it to achieve required speed and torque.

NAME: MAXON motor controllers (MAXPOS)



Fig - 6 Motor controller

These are the major components that are used in the system other components like wires, PCB boards, relay and connectors.

3. CALCULATION FOR THE SPECIFIACTIONS

• Calculation for counter weight:

By considering the Medium sized Pick-up Truck having Gross vehicle weight of almost 2300kgs.

Assumed maximum angle of slope is 20°.

The force acting on the vehicle when it is plying on the slope = mg sin θ

- = 2300*9. 8*0.39
- = 7709.13 N

On the other hand the counter balance which is located at the bottom will work in the opposite direction to the normal force. So the force component of it will be,

= mgcosθ =240*9.8*0.93 =2187.3 N

For, the 20° slope angle the counter force to balance the vehicle should be at least 20-25% of the vehicle weight and by 150 kg block the desired result can be effectively achieved.

• Calculation for motor specification:

The wight of the block is 240 kg, in order to calculate the required torque and power of the motor.

By assuming the track width distance 1.8 meter. And the block width is 0.25 meter. So both side there is a 0.65meter distance for the block to be displaced.

For torque required to move the block at the required distance is, W*d

Where W is weight of the block d is the distance from center to motor gear radius

here in this case the block is mounted on the slot of bed, but let's assume that there is no friction and the mass acts again the gravity.

In order to calculate the power of the motor,

P = Force * Velocity

- Force(F) = mass * gravitational acceleration
 - = 240 * 9.8
 - = 2352 N

P = F * v

= 2400 (~2352) * 1.5 m/s

(We required to move the block very rapidly so assume that the required velocity is 1.5 m/s)

= 3600 Nm

Since we are using two motors the power is distributes in half for each motor.

So, the required power for the motor is 2000 W for each motor. (As mentioned in the components)

4. PROGRAMMING FOR MICROCONTROLLER

Arduino is the open-source programming software for microcontroller. In order to control the both motor we need to specify the values like speed and running time in order to get the required output.

Arduino_Main_Program			
#include "I2Cdev.h" #include @ire.hp			
Finclude <pid_v1.h></pid_v1.h>			
Finclude "MPU6050_6Axis_MotionApps20.h"			
Finclude <softwareserial.h> Finclude <digitalioperformance.h></digitalioperformance.h></softwareserial.h>	//library for faster pin R/W		
//#include d/ltrasonic.h>	manual for reader particity		
Rdefine d_speed 1.5 Rdefine d_dir 3			
#define IN1 11 #define IN2 10 #define IN3 9 #define IN4 3			
char content = 'P';			
int MotorAspeed, MotorBspeed; floot MOTORSLACK_A = 40:	// Compensate for motor slock range (low PMM values which result in no motor engagement)		
float MOTORSLACK_B = 40;			
#define BALANCE_PID_MIN -255 #define BALANCE_PID_MAX 255	// Define PID limits to match PMM max in reverse and foward		
MPU6050 mpu;			
const int rxpin = 6; //Bluetooth se	erial stuff		
<pre>const int txpin = 5; SoftwareSerial blue(rxpin, txpin);</pre>			
//Ultrasonic ultrasonic(A0, A1); //int distance;			
Fig - 7 Arduino program			

VectorFloat gravity; // float ypr[3]; // double setpoint; //set th double originalSetpoint;	<pre>/ [w, x, y, z] / [x, y, z] / [yaw, pitch, roll] ilues for your 80T**** we value when the bot</pre>	quaternion container gravity vector yaw/pitch/roll container ****/ S perpendicular to ground	. ,	
<pre>double setpoint; //set th double originalSetpoint; //Read the project docume</pre>	e value when the bot			
Jeefine KU 0.6 //Set thi #define KL 0.6 //Set thi #define KL 160 //Finally #define RKd 4//Set this #define RKd 4//Set this #define RKd 4//Set this #define RKd 300 //Finally #define RKd 300 //Finally #define RKd 40/Set this #define RKd 300 //Finally define RKd 40/Set this #define RKd 40/Set thi	i first is secound r set this is first iecound y set this ing	st.com to learn how to set		
PID pid(Sinput. Soutput.	åsetpoint. Kp. Ki. Ko			
		Fig - 8 A	lrduino p	rogran

tatus vars false; // set true if DMP init was successful

Arduino_Main_Program	<u> </u>
<pre>vmid set#20 { Set all ApplicitISDMD; blow hegi(VSDMD); blow hegi(VSDMD); blow cellsmoot(20); initum(2); //mitialiser le MPU850 initum(2); //mitialiser les metures orignalistyption; = 0.1; septoirs - orignalistyption; ; yerdpoint - yertpinelSequent; ; } }</pre>	
} (Sherperty (Conc) (Sherperty (Conc) (Sherperty (Conc) (Sherperty	
<pre>vid init.tmc() { // initialize dwice // initialize dwice // initialize dwices*)); With equif(); The - 42; The - 42; </pre>	
Fig - 9 Arduino prog	gram





Fig - 10 Arduino program

5. WORKING METHODOLOGY

The components of the system are arranged as shown in the figure (Fig - 11). The gyroscope which is basically IMU with accelerometer and gyroscope, it will induce the signal and the micro controller executes some command to the motor controller after reading signals. The motor controller rotates the motor at required torque and speed which are calculated by the microcontroller.

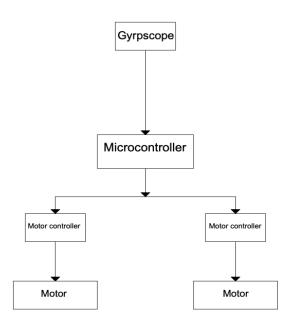


Fig - 11 Working

When the slope is on the left side the centripetal forces will act on the vehicle in the right direction, and that causes the vehicle to roll-over. But this movement will detect by the gyro sensors and a specific output will be generate and the signals are feed to the microcontroller. Microcontroller then will generate the controlling signals for the motor and feed it to the motor controller and it will control the motor such that the motor rotate in the appropriate speed and generate the specific torque so the required displacement of the block is archived on the left side.

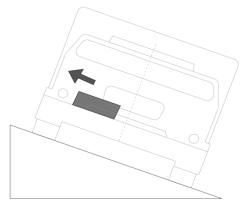


Fig - 12 Left slope

Same process for the right side slope in which the block will move to the right in order to prevent the roll-over at the left.

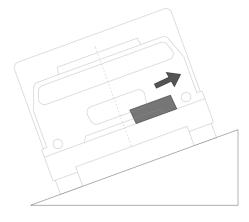


Fig - 13 Right slope

6. CONCLUSION

The system works effectively with given programming and components. This system is beneficial for industry and government as well because it increases the safety of the driver and stakeholders. However it works for the limited angle set and system become unstable during the power cut-off. For the very heavy vehicle the system requires a heavy block which is economically impractical as it increases the gross vehicle weight and hence the fuel consumption.

REFERENCES

- [1] M. Hollenbeck, H. Weinblatt, and N. C. H. R., Equipment for Traffic Load Data. Transportation Research Board, 2004.
- [2] R. White, J. Song, C. Haas, and D. Middleton, "Evaluation of Quartz Piezoelectric Weighin-Motion Sensors," Transportation Research Record: Journal of the Transportation Research Board, vol. 1945, pp. 109–117, Jan. 2006, doi: 10.3141/1945-13.
- [3] B. Jacob, WAVE: Weigh-in-motion of Axles and Vehicles for Europe. 2002.

- [4] Taylor R. (1990), Interpretation of the Correlation Coefficient: A Basic Review, JDMS 1, January/February 1990, pp. 35-39,
- [5] Hajian, M., Arab Markadeh, G. R., Soltani, J., & Hoseinnia, S. (2009). Energy optimized sliding-mode control of sensorless induction motor drives. Energy Conversion and Management, 50, 2296–2306.
- [6] aoré, D., Leon, D., & Glumineau, A. (2012). Adaptive interconnect observer-based backstepping control design for sensorless induction motor. Automatica, 48, 682–687.

BIOGRAPHIES



Mahmmadabararkhan N. Diwan Automobile Engineering, Gujarat Technological University (2017-2021).



Dhruv R. Patel Automobile Engineering, Gujarat Technological University (2017-2021).



Dhairyay U. Patel Automobile Engineering, Gujarat Technological University (2018-Current).



Kritik M. Patel Mechanical Engineering, Gujarat Technological University (2018-Current).

Divyesh M. Rane Mechanical Engineering, Gujarat Technological University (2018-Current).