

Cost Effective Ball Milling Machine for Producing Nanopowder

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Abstract - Research work related to metallurgy and mechanical department deals with the study of various ferrous and nonferrous materials. Usually properties of these materials enhance at microscopic level. These changes in properties have numerous applications. But in order to carry out research activities there must be ample supply of Nanopowder at low cost. Unfortunately this is not the case. The Nanopowder now obtained is produced through high energy planetary ball milling machine. The cost of this machine is high. Also the amount of powder produced is more. Our effort in this project is to minimize the production cost of Nanopowder by providing alternate machine. The major advantages of this machine include lower floor space, low initial cost. Also the quantity of nano powder required for testing is usually less. So the above mentioned machine can be used effectively

Key Words: Ball milling, Cost effective, Nanopowder

1. INTRODUCTION

Nanotechnology is the science that deals with matter at the scale of 1 billionth of a meter (i.e., 10^{-9} m=1nm), and is also the study of manipulating matter at the atomic and Molecular scale. A nanoparticle is the most fundamental component in the fabrication of a nanostructure, and is far smaller than the world of everyday objects that are described by Newton's laws of motion, but bigger than an atom or a simple molecule that are governed by quantum mechanics. In general, the size of a nanoparticle spans the range between 1 and 100 nm.

There are several Synthesis methods for creating nanoparticles like, Physical vapor deposition, Chemical vapor deposition, Sol-gel Method, RF Plasma Method, Pulsed Laser Method, Thermolysis and Solution Combustion Method. But these methods are Very costly, so comparatively cheaper method has to be developed.

Synthesis of nanomaterials by a simple, low cost and in high yield has been a great challenge since the very early development of nanoscience. Various different processes have been developed for the commercial production of nanomaterials. Among all top down approaches, high energy

ball milling, has been widely exploited for the synthesis of various nanomaterials.

Various types of high-energy milling equipment are used to produce nanopowder. They differ in their capacity, efficiency of milling and additional arrangements for cooling, heating, etc. 1. SPEX shaker mills 2. Planetary ball mills 3. Attritor mills. These methods although cheaper than the chemical method but still are very high priced. Hence in this project we have developed a machine which will manufacture nanopowder cheaper than the above mentioned methods. A principle of high-energy ball milling is used in order to produce nanopowder. Our project is beneficial to most of the educational institutions to carry out research work related to nanotechnology. It is a simple process and does not include working with hazardous chemicals.

2. DESIGN

I. DESIGN

Shaft Design:

Motor used = 1HP

So Power P = (kw) = 0.735 KW, Speed (N) =750rpm.

Tension on tight side and slack side of the belt:

$$V = \frac{\pi \times 40 \times 750}{60} = 1.57 \text{ m/s. } D = \text{Diameter of Pulley.}$$

$$\text{So power (P)} = (T_1 - T_2) V$$

$$735 = (T_1 - T_2) \times 1.57$$

$$T_1 - T_2 = 468 \text{ N}$$

$$\text{Angle of Wrap } \theta = \pi - 2 \sin^{-1} \left(\frac{70 - 30}{708} \right) = 3.02 \text{ radians}$$

We know that,

$$\frac{T_1}{T_2} = e^{\mu \theta} = e^{(0.27 \times 3.02)}$$

$$T_1 = 2.26 T_2$$

$$2.26 T_1 - T_2 = 468 \text{ N}$$

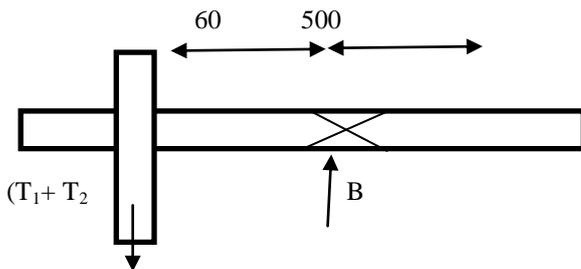
$$T_2 = 371.42 = 371 \text{ N.} \quad T_1 = 839 \text{ N} = 840 \text{ N}$$

Weight of pulley $W = 0.75 \times 9.81 = 7.5 \text{ N.}$

Torque available on shaft,

$$Mt = \frac{60 \times 10^6 \times (KW)}{2\pi \times N} \text{ N} = \text{minimum Speed available on shaft.}$$

$$Mt = \frac{60 \times 10^6 \times 0.725}{2\pi \times 200} = 35093.66 \text{ Nmm.}$$



Downward Force = $T_1 + T_2 = 1310 \text{ N.}$

So Moment @ C = $R_B \times 500 - 1310 \times 500 \text{ Nmm}$

$$R_B = 1467.2 \text{ N}$$

Moment @ B = $(1310) \times 60 = 78600 \text{ Nmm}$

Failure criteria for shaft (Torsional & Bending)

$$\tau_{max} = \frac{16}{\pi d^3} \sqrt{(78600)^2 + (35093.66)^2}$$

$$\frac{390 \times 0.5}{3} = \frac{16}{\pi d^3} \sqrt{(78600)^2 + (35093.66)^2}$$

$$C = \frac{5_4 t \times 0.5}{F (s)}$$

$d = 19.05 \text{ mm, so } d = 20 \text{ mm.}$

Voil Design

From design data book / metallurgy book

Density of graphite powder: 2160 kg/m^3

Density of strain hardening steel ball = 7850 kg/m^3

The voil is designed for milling of 250 gm. of graphite material

So, Mass of the graphite material = 250 gm.

So we know that,

Mass (m) = density (ρ) \times volume

$$\text{So } V = \frac{m}{\rho} = \frac{0.250}{2160} = 1.1574 \times 10^{-4} \text{ m}^3$$

Volume of Graphite material (V_g) = $1.1574 \times 10^{-4} \text{ m}^3$

(ii) Mass of the ball = 10 times of the graphite material

So (M)b = $10 \times (M)g = 10 \times 250 = 2500 \text{ gram} = 2.5 \text{ kg}$

$$\text{So volume of ball} = \frac{(m)b}{(\rho)b} = \frac{2.5}{7850} = 3.1847 \times 10^{-4} \text{ m}^3$$

(iii) Volume of the blade, (V)_{blade} = $l \times b \times h$

$$(V)_{\text{Blade}} = (0.03) \times (0.005) (0.01) \times 4 = 6 \times 10^{-6} \text{ m}^3$$

(iv) Volume of shaft, (V)_s = $\frac{\pi}{4} \times d^2 \times (1.25d)$

$$= \frac{\pi}{4} \times (0.02)^2 \times (1.25 \times 0.02) = 7.8539 \times 10^{-6} \text{ m}^3$$

$$(V)_{\text{Material}} = (V)_{g} + (V)_{b} + (V)_{\text{Blade}}$$

$$= 1.1574 \times 10^{-4} + 3.1847 \times 10^{-4} + 6 \times 10^{-6}$$

$$= 4.4021 \times 10^{-4} \text{ m}^3$$

Now, (Free Space = $2 \times (V)_{\text{Material}}$)

Total Volume = $2 \times (V)_{\text{Material}}$

$$= 2 \times 4.4021 \times 10^{-4} = 8.0842 \times 10^{-4} \text{ m}^3$$

Volume of Voil = Total Volume + Shaft Volume

$$= 8.0842 \times 10^{-4} + 7.8539 \times 10^{-6}$$

$$= 8.1627 \times 10^{-4} \text{ m}^3$$

Volume of Voil = $\frac{\pi}{4} \times (D_i)^2 \times (1.25D_i)$

$$8.1627 \times 10^{-4} = \frac{\pi}{4} \times 1.25 \times D_i^3$$

$$D_i^3 = 8.3144 \times 10^{-4} \text{ mm}^3$$

$$D_i = 0.09403 \text{ m} \quad D_i = 94.03 \text{ mm} = 100 \text{ mm}$$

So, Internal Diameter of Voil = 100 mm

Thickness of Voil = 10 mm Outer Dia. Of Voil = 120 mm.

Bearing Design

Selection of Bearing at Shaft,

Radial Force (F_r) = 1310 N Axial Force (F_a) = 705 N

Speed = 750 RPM/ Bearing Life = 12000 Hrs.

Diameter of Shaft (d) = 20 mm.

$$\text{So, } \frac{F_a}{F_r} = \frac{7.5}{1310} = 6.19 \times 10^{-3} < e$$

Factor X=1, Y=0,

So Equivalent Dynamic Load (P) = $X F_r + Y F_a$

$$P = X F_a P = 1310 \text{ N}$$

$$\text{Rated Bearing Life } (L_{10}) = \frac{60 \times n \times L_{10} h}{10000000}$$

$$= \frac{60 \times 750 \times 12000}{10000000} = 54 \text{ Million Revolutions}$$

Now, Dynamic Load Capacity,

$$C = P(L_{10})^{\frac{1}{3}} = 4573.51 \text{ N}$$

From Table (V.B Bhandari)

For Diameter (d) = 20 mm

Bearing = 16404 (L= 7020 N)

d = 20 mm D = 40 mm B = 8 mm.

3. CONSTRUCTION

For our experiment, we used EN8 (Carbon 0.36-0.44%, Silicon 0.10-0.40%, Manganese 0.60-1.00%, Sulphur 0.050 Max, Phosphorus 0.050%) as our grinding media and grinding jar. For the jar, inner diameter of 100 mm and thickness of 10 mm is chosen as our design. The jar has approximately $8.1627 \times 10^{-4} \text{ m}^3$ of volume and it can withstand $4.4021 \times 10^{-4} \text{ m}^3$ of grinding media. The voil (grinding jar) cap has an arrangement that clamps itself with voil. The voil cap has an opening for the shaft. The shaft of diameter 25.4 mm is inserted through this opening. The blades are attached to the shaft with the help of grab screw. For grinding media, 5mm and 10mm diameter of stainless steel balls are used. For our shafts, we used the mild steel provided in store and went to labs for machining purpose.



Fig no 1: Voil

There are three shafts in our system- main shaft and speed reduction gearbox shaft. The head of speed reduction gearbox shaft is reduced to 14mm diameter using lathe machine to fit in the pulley which has 16mm hole at middle. For the body of shaft, we reduced it into 20mm diameter.

The shaft has two universal joints to compensate for the angular movement when the shaft swivels.



Fig no. 2: Main Shaft

The upper side of the shaft is connected to the pedestal bearing. Due to the bearings have hole of 17mm; we make a small portion of 16.9- 16.95mm on shaft to fit in the bearing since the bearing has small tolerance. Four speed pulley is attached at one end of the shaft. The other pulley is mounted on the motor shaft. A V-belt is provided which takes care of power transmission.



Fig no 3: Pulley

The motor is clamped vertically to the frame using bolts. Also adequate support is provided to the motor. The motor has operating speed of 690 rpm. The motor also drives another pulley. This power will be used to swivel the voil at very low speed. During the actual working conditions it is necessary to swivel the voil because there is a definite chance that the powder will adhere the edges and if the temperature rises certain chemical bonding may take place. Swiveling also guarantees continuous processing of powder. The speed reduction is obtained using various mechanisms notably difference in diameter of pulleys, speed reduction gearbox and rack and pinion. All this arrangement is placed on the base plate which is 5 mm thick and 2.5 feet to 2.2 feet in cross section. It is also elevated from the base at height of

about 60 mm motor drives another pulley. This pulley is directly connected to the shaft of speed reduction gearbox. The speed reduction gearbox has the reduction ratio of 15:1.



Fig no 4: Voil Handle

The drive from the speed reduction gearbox is given to the voil through rack and pinion. The rack and pinion is used to provide to and fro motion to the voil. The rack is placed on the voil handle. The voil handle is designed for easy removal of voil whenever required. Bearing are provided at each side for proper functioning. These bearings are attached to main frame which is made up of M.S square bar 1.5*1.5 inches in cross section.



Fig no 5: Machine

4 WORKING

The three phase A.C supply is given to the motor. The motor has power of 1 Hp. The motor is clamped vertically to the frame using bolts. The shaft of motor has two pulleys. viz, four speed pulley and single speed pulley. The power is transmitted from four speeds from four speed pulley to the main shaft. The main shaft which has two universal joints is fixed with the help of bearings. The maximum speed achieved in this case is 690 rpm. This shaft has four blades attached to its end using grab screw. The shaft is then placed inside the voil. Suitable amount of ball bearings are placed inside the voil. During the actual operation, due to high speed ball milling principle actual powder is formed. Suitable time gaps are provided so that the powder does not adhere to the face of voil.

This is intermittent process and takes about 3-4 hours as per the volume of solid to be powdered. The voil has a separate clamping mechanism. The voil is held in the voil handle. The voil handle is designed for easy removal of voil whenever required. The voil handle moves to and fro by rack and pinion mechanism. This mechanism is attached to the separate frame which is welded to base. The rack slides up and down through a plate which is attached to the speed reduction gearbox shaft. The power input from the motor is given to the speed reduction gearbox through pulley

5 RESULTS

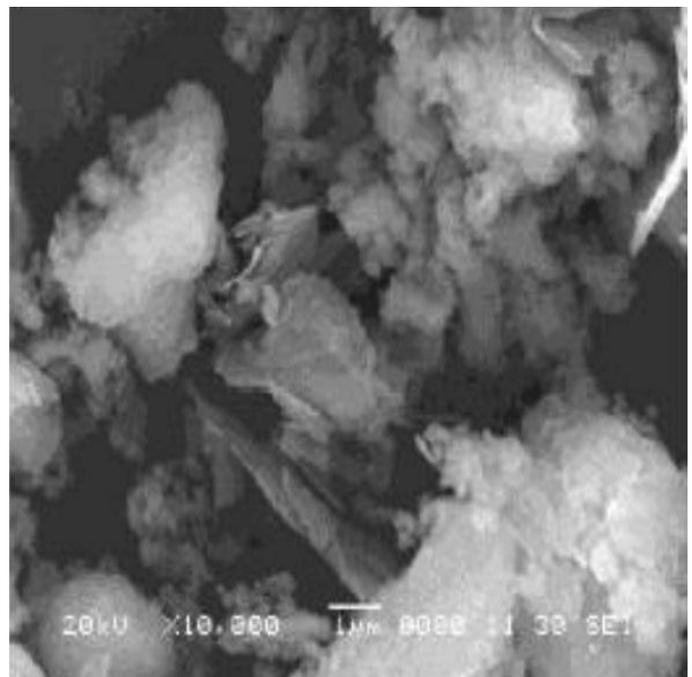


Fig no 6: SEM image of graphite after 100 min milling with SS balls



Fig no 7: SEM image of graphite after 200 min milling with SS balls

The powder obtained from the machine was tested on scanning electron microscope (SEM). Figure 6 shows image of graphite after 100 min milling with SS balls and Figure 7 shows image of graphite after 100 min milling with SS balls. When tested their size was within the range of nanoparticles. Thus it can be concluded that this machine can produce nanopowder effectively.

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