Design and Analysis of Fixture for component cradle in HMC-800

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Abstract - Main objective of project work is to design a fixture which can used with horizontal machine center 800, for manufacturing a cradle component. Here fixture is designed such that it should hold, support and guide the cradle component, where different machining operation is carried out. Firstly individual component of fixture is designed and assembly is done in solid works. ANSYS software is used for evaluation of stress and displacement of critical parts and same parts are also analyzed by analytical method. Finally comparative study is carried out.

Key words: Horizontal Machining Centre-800, Solid works, and Ansys Software.

1. INTRODUCTION

Now a day, the small scale industries are trying to increase the demand of the product and increase the mass production. To meet these challenges it has become very important for companies to increase their production rate. The successful running of any mass production depends upon the interchangeability of the work parts to facilitate easy assembly and reduction of unit cost. Mass production demands a fast and easy method of positioning work for accurate operation on it. The main intention of any company is to provide good quality product and increase in production rate in order to get profit over it. This can be achieved by minimizing manufacturing lead time and cost of production by using work holding guiding device. This work holding device is fixture. Fixture word is derived from latin word which means fixed or attached.

Fixtures are designed such that large number of components can be machined or assembled identically. Fixtures are special purpose tools which are used to facilitate production when work piece are to be produced in mass production scale. Advantages of fixtures are, once the fixture is properly setup, any number of duplicate parts can be readily produced without any additional setup. Hence increasing the productivity and increasing accuracy by reducing setup cost and manual fatigue.

It is a device for locating, holding and supporting a work piece during a Manufacturing operation. Fixtures are essential elements of production processes as they are required in most of the automated manufacturing, inspection, and assembly operations.

1.1 Purpose of Fixture

The main purpose of the fixture is to locate the work piece quickly and accurate, support it proper and hold it securely, thus ensures that all parts produced in the fixture will come out alike within the specified limits. In this way accuracy and inter changeability of the parts are provided.

By maintaining or even improving the inter changeability of the components, the fixtures help to significantly reduce assembly costs, maintenance costs, and the potential of subsequent standard machines, and the quality of the parts. An important goal is to design a fixture that makes it very convenient and adds security to the operator as well as to the machine and other used components.

1.2 Classification of Fixtures

Fixtures are classified according to following factors.

Application Based
1. Work holding fixtures
2. Tool holding fixtures
3. Fit- up fixtures
4. Gauging fixtures
5. Holding fixtures

Based On Degree of Mechanization and Automation
1. Hand operated
2. Power operated
3. Semi-automatic
4. Automatic

Based On Operation
1. Milling fixtures
2. Drilling fixtures
3. Turning fixtures
4. Grinding fixtures
5. Broaching fixtures
6. Welding fixtures
7. Assembly fixtures

1.3 Elements of Fixtures

Generally, all fixtures consist of the following elements:
a) Locators b) Clamp c) Supports d) Fixture body
1.4 Materials Used In Fixture

The materials used in fixture are –

1. **High-speed steel:** Mainly used for cutting tools, such as drill bits, reamers and Cutting tool. These can be oil hardened to 64 to 66 Rc, contains about 1% carbon, 0.5 to 1.5% tungsten and a small amount of silicon, chromium, molybdenum and less carbon. Molybdenum and so on.

2. **Die steel:** These are mainly used for hot working or cold working moulds, die steel for stamping tools that can be hardened to 65RC, which contains about 1% carbon, 0.5 to 1.5% tungsten and a small amount of silicon. Oil steel hardened to 60 RC, containing 2-5% chromium, 4-9% tungsten, 0.4-0.5% vanadium and a smaller amount of manganese and silicon.

3. **Carbon steel:** These can be used for standard cutting tools. They contain 0.85% carbon, 0.5-0.8% manganese and a smaller amount of silicon. These can be hydrated to 62 to 63% of the RC. The steel can be used for drill bushings, locators and other parts that are worn and require hardening.

4. **Cast Iron:** Used for processing and piecing the odd shape when manufacturing, CI need to use casting mode. The cost of the model should be compared with the processing and manufacturing costs. Iron content of more than 2%, can withstand vibration is very suitable for milling base and body. The self-lubricating property of castiron is suitable for machine rails.

5. **Non Shrinking tool steels:** This is also known as high carbon (1-2%) or high chromium (4-12%) steel. These steels are distorted during heat treatment and are easy to handle when they are not possible to eliminate heat treatment distortion. Oil is hardened to 60-64 RC. These steels are widely used in fine, complex pressure tools.

1.5 Objective

The objective of this project is

- Design and manufacturing of individual parts and assembly of fixture for a cradle component.
- Fixture is designed such that it is only suitable for horizontal machining Centre (HMC-800).
- Initial design of a fixture is done as per require size of cradle component.
- The fixture has to be designed for holding and guiding the cradle component while performing the drilling, facing, milling, tapping operation on it.
- Analytical calculation of fixture design is compared with numerical values.

1.6 Methodology

- Study of the cradle component and the existing fixture in use.
- Fixture planning: Estimation of the process for achieving the final dimensions of the components, analyzing all the available information regarding the material, geometry of the cradle component and operations required.
- Fixture layout is to represent the fixture concepts in a physical form. i.e. Positions of locators, Positions of clamps, Positions of supports and sequence.
- Fixture element design: The fixture design has been done using modeling software solid works.
- Fixture assembly design: after completion of designing individual part of the fixture, fixture assembly has been carried out.
- After completion of fixture design, casting and machining has been carried out for all the individual parts.
- Using the machining parameters, the force exerted on the component is calculated. The analytical formulas are used to calculate stress and deformation for critical parts.
- Analysis of critical components: The static analysis is carried out by finite element method using ANSYS software to find out stress and deformation.
- Analytical results for critical parts are validated with FE analysis and found good results.

2. LITERATURE REVIEW

**N. P. Maniar and D. P. Vakharia,** HMC is best solution for particular component but designer can’t ask industries to replace CNC with HMC because of the cost factor, as HMC cost more than CNC. With the help of creo element/proe5.0 the unbalance mass and its location of C.G are found out and it is remarkably same as experimental result on dynamic balancing machine. So, Computer aided mass balancing of quadrants is found more accurate to decrease in percentage error by almost 6%. [1]

**C. Radha Madhavi, K. Srinivasulu et al** this project presents the design of fixture for gas turbine rotor blade for machining on VMC (Vertical Machining Centre). This report consists of study of input data from customers like part drawing and assembly drawing. The fixture design begins with part modeling, machining and analysis of various parts in the fixture assembly using solidworks, for an analysis COSMOS software package is used. [2]

**Dr.Yadavalli Basavaraj and Pavankumar B K** proposed about modeling and analysis of support pin for barke spider fixture by FEM using ANSYS. In this detailed study of Brake spider is carried out by using CATIA V5 modeling software and it is evaluated for failure of support pin component by FEM using ANSYS. This modified design is adapted in
fabrication of fixture and is tested for its productivity. From static analysis maximum cutting force and maximum clamping force are found. Maximum static deflection and maximum stress are also found. [3]

P. Satish Reddy and V. Subrahmanyam In this paper, the design requirements of the fixture were studied, customers are using horizontal machining center for parts processing. Their concept is to place the components vertically on the machine. To this end, they choose the welding structure. Complete the analysis of the clamp to check whether the fixture is subjected to the maximum cutting force during machining. [4]

3. DETAILS OF Fixture FOR COMPONENT CRADLE

Solid work has been used to sketch 2D daigram of fixture for component cradle. Later on 2D sketch is converted into 3D modelling in the solid work and assembly has been done. Individual parts of fixture is given below.

<table>
<thead>
<tr>
<th>1. Base Plate</th>
<th>9. Heel Pin</th>
</tr>
</thead>
<tbody>
<tr>
<td>2. Rest Pad</td>
<td>10. Special Washer</td>
</tr>
<tr>
<td>3. Supporting Block-1</td>
<td>11. Conical seat</td>
</tr>
<tr>
<td>7. Special Stud</td>
<td>15. Boss</td>
</tr>
<tr>
<td>8. Special Hex Bolt</td>
<td></td>
</tr>
</tbody>
</table>

Table 1: Specification of drilling operation

<table>
<thead>
<tr>
<th>Speed (m/min)</th>
<th>2100 RPM</th>
</tr>
</thead>
<tbody>
<tr>
<td>Drill Diameter</td>
<td>Ø13mm</td>
</tr>
<tr>
<td>Feed Per Revolutions</td>
<td>0.05-0.06 mm/Rev</td>
</tr>
</tbody>
</table>

3.1 Analytical Calculation and FE Analysis
3.2 Force calculations for Drilling:

Following specification of drilling operation are used to calculate the cutting force in HMC machine.

<table>
<thead>
<tr>
<th>Speed (m/min)</th>
<th>2100 RPM</th>
</tr>
</thead>
<tbody>
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<td>Ø13mm</td>
</tr>
<tr>
<td>Feed Per Revolutions</td>
<td>0.05-0.06 mm/Rev</td>
</tr>
</tbody>
</table>

Figure: 2 Horizontal Machining Center-800

Cutting Speed: (m/min)

\[ V = \pi \frac{D n}{1000} \]
\[ V = (\pi \times 13 \times 2100)/1000 \]
\[ V = 85.76 \]

Power at the Spindle: (KW)

\[ N = 1.25 \times D^2 \times k \times n \left(0.056 + 1.5 \times s\right) /10^5 \]
\[ = 1.25 \times (13)^2 \times 2100 \times 2.1 \times (0.056+1.5 \times 0.05)/10^5 \]
\[ N = 1.222 \]

Power at the Motor: (KW)

\[ N_{el} = \frac{N}{E} \]
\[ = 0.1220/0.8 \]
\[ N_{el} = 1.525 \]

Torque at Spindle: (N-m)

\[ Ts = 975 \times N/n \]
\[ = (975 \times 1.220) / 2100 \]
\[ Ts = 0.566 \]
Cutting Force: (N)

\[ T_f = 1.16 \times K \times D \left( 100 \times s \right)^{0.05} \]
\[ = 1.16 \times 2.1 \times 13 \left( 100 \times 0.2 \right)^{0.05} \]
\[ T_f = 1243.78 \]

Analytical Calculation For support Pin

Force \( F = 1243N \)

<table>
<thead>
<tr>
<th>Area: (mm(^2))</th>
<th>Deformation: (mm)</th>
</tr>
</thead>
<tbody>
<tr>
<td>( A_1 = \pi d^2/4 )</td>
<td>( \delta = FL/AE )</td>
</tr>
<tr>
<td>( = \pi 35^2/4 )</td>
<td>( = 962.11 )</td>
</tr>
<tr>
<td>( = \pi 20^2/4 )</td>
<td>( = 314.15 )</td>
</tr>
<tr>
<td>( = \pi 40^2/4 )</td>
<td>( = 1256.6 )</td>
</tr>
<tr>
<td>( A_T = 2532.86 )</td>
<td>( \delta = 2.14 \times 10^{-4} )</td>
</tr>
</tbody>
</table>

Stress: (MPa)

\[ \sigma = F/A \]
\[ \sigma = F \left[ \left( 1/A_1 \right) + \left( 1/A_2 \right) + \left( 1/A_3 \right) \right] \]
\[ \sigma = 400 \left[ \left( 1/962.11 \right) + \left( 1/314.15 \right) + \left( 1/1256.6 \right) \right] \]
\[ \sigma = 2 \]

3.3 FE ANALYSIS

In the static structural analysis, the displacements and stresses, due to external force, are determined. There are two kinds of structural analysis; one is linear structural analysis and another is non-linear structural analysis.

3.4 Material Properties

For the fixture, the material considered in the analysis is Steel. The mechanical properties of the Steel are shown in below.

<table>
<thead>
<tr>
<th>Properties</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Young’s modulus</td>
<td>2.0E+05 MPa</td>
</tr>
<tr>
<td>Poisson’s ratio</td>
<td>0.3</td>
</tr>
<tr>
<td>Density</td>
<td>7850 kg/m(^3)</td>
</tr>
<tr>
<td>Yield strength</td>
<td>250 MPa</td>
</tr>
<tr>
<td>Tensile Ultimate strength</td>
<td>400 MPa</td>
</tr>
</tbody>
</table>

Table 2: material properties of steel

Support pin:

The model is constrained at the bottom surface of base structure and force is applied at the top portion as shown in below figure.

The displacement of support pin is shown in above figure. The maximum static deflection of support pin is found to be 3.0092 \( \times 10^{-4} \) mm.

The stress of support pin is shown in above figure. The maximum stress of support pin is found to be 2.2896 Mpa.
Figure 6: von-Mises stress plot

The von-Mises stress of support pin is shown in figure 30. The maximum von-Mises stress of support pin is found to be 7.155 Mpa.

<table>
<thead>
<tr>
<th>Component Name</th>
<th>Theoretical Results</th>
<th>FEA Results</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Deflection (mm)</td>
<td>Stress (Mpa)</td>
</tr>
<tr>
<td>Support Block-1</td>
<td>1.51 × 10^{-4}</td>
<td>1.559</td>
</tr>
<tr>
<td>Support Pin</td>
<td>2.14 × 10^{-4}</td>
<td>2.01</td>
</tr>
<tr>
<td>Support Block-2</td>
<td>4.636 × 10^{-5}</td>
<td>0.238</td>
</tr>
</tbody>
</table>

Table 3: Comparison of Theoretical and FEA Results

4. CONCLUSION

- The fixture for cradle component has been designed successfully, in order to increase the accuracy and productivity.
- The cutting forces were calculated for all machining operation. The maximum cutting force in drilling operation i.e. 1250N.
- The static analysis is carried out by finite element method using ANSYS software.
- Analytical results for critical parts are validated with FE Analysis and found relatively good results.
- The maximum static deflection, stress and von-Mises stress is found in support block-1 i.e. 1.834 × 10^{-4} mm, 1.08 MPa and 3.1608 MPa. These values are within the design limits.
- The maximum static deflection, stress and von-Mises stress is found in support pin i.e. 3.0092 × 10^{-4} mm, 2.289 MPa and 7.155 MPa. These values are within the design limits.
- The maximum static deflection, stress and von-Mises stress is found in support block-2 i.e. 5.6 × 10^{-5} mm, 0.3215 MPa and 3.08 MPa. These values are within the design limits.

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REFERENCES