

ESTIMATION OF NATURAL FREQUENCY OF MOTORCYCLE HANDLEBAR

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Abstract - This paper gives the natural frequency motorcycle handlebar (HERO SPLENDOR PLUS). The motorcycle handlebar is modeled by using CATIA and analyzed by using ANSYS a conventional FEM package. Here we are going to compare the natural frequency by FEM package with the practically by using FFT analyzer, so as to compare working frequency with natural frequency for validation purpose.

Key Words: Natural frequency HERO SPLENDOR PLUS, handlebar, ANSYS, FFT analyser

1.INTRODUCTION

Motorcycle handlebar is rider's fist touch point to the motorcycle. It is complex in structure and has important role as functionality and safety point of view. Handlebar assembly consists of mirrors, brake levers, clutch, head lamp and plastic coverings. The failure comes in the handlebar because of engine vibrations, uneven road surface, bumps, rider force, braking etc.

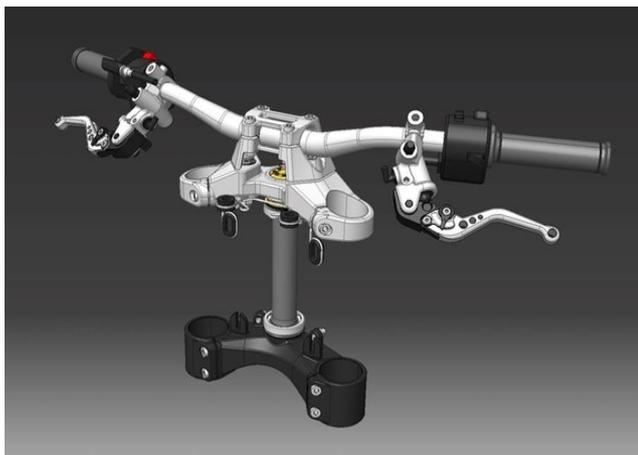


Fig-1: Handlebar assembly



Fig- 2: Motorcycle handlebar (HERO SPLENDOR PLUS)

2. ESTIMATION OF NATURAL FREQUENCY

The natural frequency of a system is the frequency at which it will vibrate freely (i.e in the absence of damping) in simple harmonic motion, when set in motion. Free vibrations of any elastic body are called natural vibrations and occur at a frequency called the natural frequency. The natural frequency of a structure depends on its physical characteristics like for example - its mass and stiffness. A system with n degrees-of-freedom, will have n modes of vibration - each of which has its own natural frequency.

2.1 Mathematical Modeling

Mathematically Natural Frequency of the handlebar is calculated as below

$$\omega_n = \sqrt{\frac{k}{m}} \dots\dots (1)$$

where k is stiffness of handlebar

$$k = \frac{A.E}{L}$$

$$A = \frac{\pi}{4} (d_o^2 - d_i^2)$$

where d_o=outer diameter of handlebar
=22.3mm

d_i=inner diameter of handlebar
=18.4

L=length of spring

L =1000 mm

E =Young’s modulus

E =200 GPa

and m is mass of spring,

m= ρAL

where ρ is density of the handlebar material

ρ=7860 ×10⁻⁹ kg/mm³

$$\omega_n = \sqrt{\frac{AE}{L\rho AL}}$$

$$\omega_n = \sqrt{\frac{E}{\rho L^2}}$$

$$\omega_n = \sqrt{\frac{210 \times 10^3}{7860 \times 10^{-9} \times 1000^2}}$$

$$\omega_n = 163.46 \text{ Hz}$$

Hence natural frequency of the handlebar found out by mathematical calculation is $\omega_n = 163.46 \text{ Hz}$.

2.2 Finite Element Analysis of HERO SPLENDOR PLUS Handlebar

The finite element analysis of HERO SPLENDOR PLUS handlebar is done by Ansys. Boundary conditions are applied as shown in fig 3.

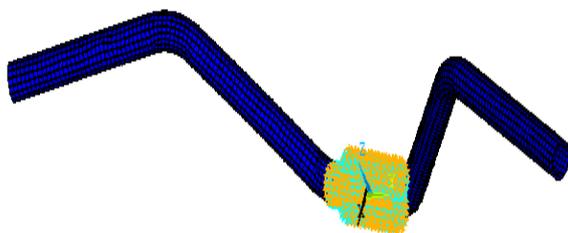


Fig-3: Boundary condition

The fig. 4 shows the natural frequency ($\omega_n=137.343 \text{ Hz}$). It is the first mode of natural frequency.

DISPLACEMENT
STEP=1
SUB =1
FREQ=137.343
DMX =98.4257

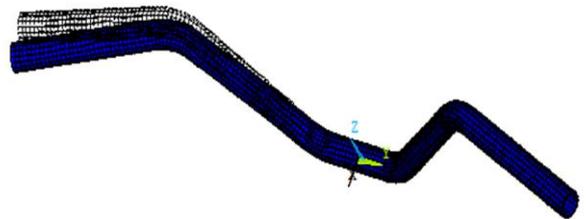


Fig- 4: Natural frequency (1st Mode)

The fig. 5 shows the natural frequency ($\omega_n=137.347 \text{ Hz}$). It is the second mode of natural frequency.

DISPLACEMENT
STEP=1
SUB =2
FREQ=137.347
DMX =98.4309

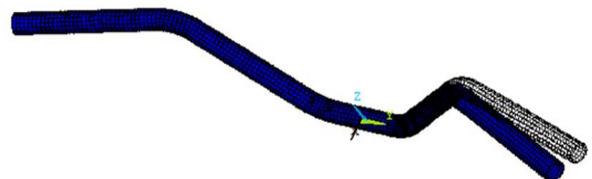


Fig- 5: Natural frequency (2nd Mode)

The fig. 6 shows the natural frequency ($\omega_n=140.444 \text{ Hz}$). It is the third mode of natural frequency.

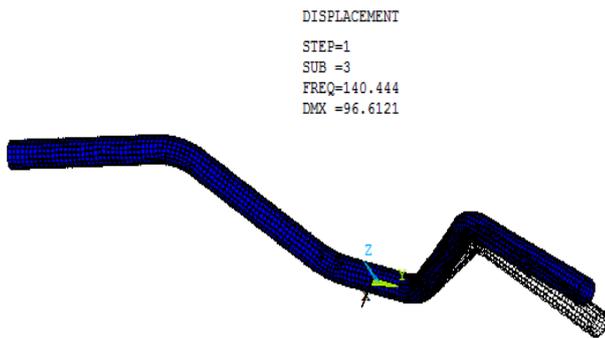


Fig- 6: Natural frequency (3rd Mode)

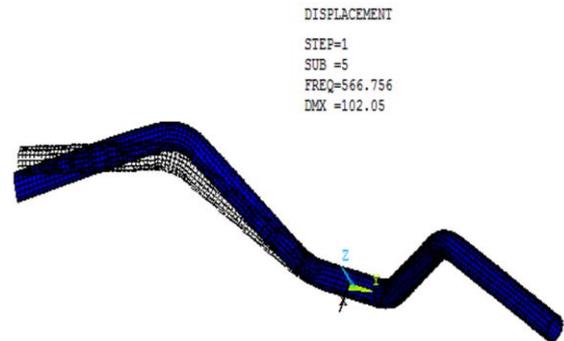


Fig-8: Natural frequency (5th Mode)

First five natural frequencies of handlebar are given in tabular form as below

The fig. 7 shows the natural frequency ($\omega_n=140.459$ Hz). It is the fourth mode of natural frequency.

Table 1. First Five Natural Frequencies of Handlebar

Sr. No.	Mode Number	Natural Frequency (Hz)
1	1	137.343
2	2	137.347
3	3	140.444
4	4	140.459
5	5	566.756

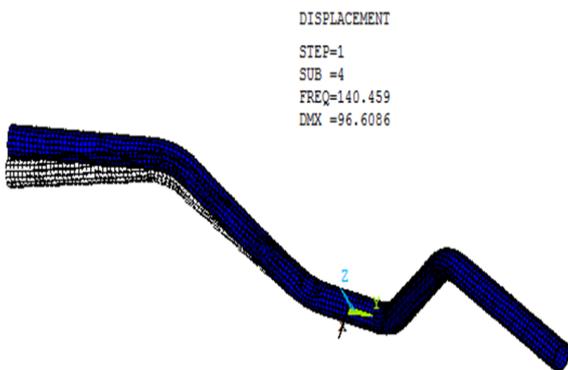


Fig-7: Natural frequency (4th Mode)

The fig. 8 shows the natural frequency ($\omega_n=566.756$ Hz). It is the fifth mode of natural frequency.

2.3 Experimental Validation

There are two requirements for the use of computational tools for simulations: verification and validation. It doesn't matter how high the computational power of your system is or what computational method you are using, the result obtained is still an approximation. And there are certain cases where these approximations fail. (e.g. when stress distribution has a concave shape and high magnitude). For any proposed problem the desired result is always real world implementation, therefore experimental validation though expensive, is essential. In this experimentation handlebar is in the hanging position which is hanged by using two strings at two free ends i.e. the handlebar is in free free condition. The accelerometer is mounted at middle of the handlebar. Four channel FFT

analyser is attached to it. Handlebar is struck by feedback hammer.



Fig- 9: Experimental validation of natural frequency

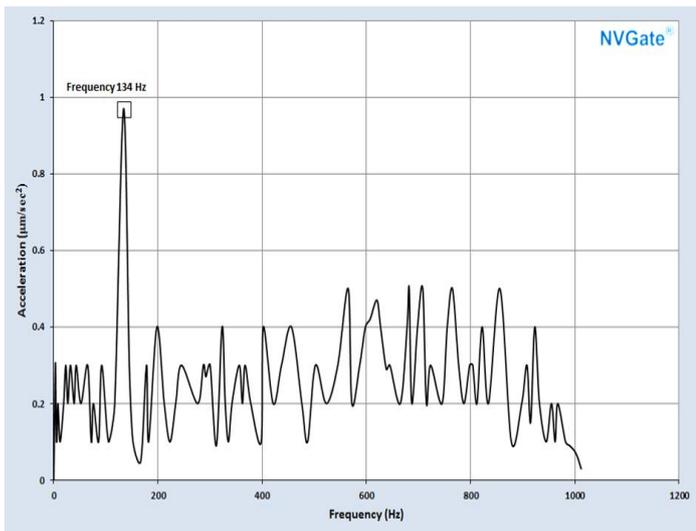


Fig-10:Natural frequency (Experimental)

The resulting natural frequencies are acquired by using NV Gate software as shown in figure 10 which shows that the natural frequency is 134 Hz.

3. CONCLUSION

The natural frequency is calculated by using three different methods such as mathematical calculation, finite element analysis and experimental methods.

- Natural frequency of the handlebar found out by mathematical calculation is $\omega_n = 163.46 \text{ Hz}$.

- Natural frequency found out by FEA for the 1st mode is $\omega_n = 137.343 \text{ Hz}$.
- Natural frequency found out by experimental test is $\omega_n = 134 \text{ Hz}$.

The results are compared and found to be approximately same.

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REFERENCES

- [1] S. S. Rao, Mechanical Vibration, 3rd ed., McGraw-Hill Publisher, New Delhi.
- [2] Cyril M. Harris, Allan G. Piersol, Shock and vibration handbook, 5th ed., McGraw-Hill Publisher, 2002
- [3] Amol H. Parihar, Pravin P. Huzare, Swapnil S. Kulkarni, Determination of natural frequencies of an handle bar using finite element methods to enhance the strength, International Journal of Scientific Research and Management Studies (IJSRMS), Volume 1, Issue 11, pp-346-351.
- [4] S. Agostoni, F. Cheli, E. Leo, M. Pezzola, An

innovative multi dof TMD system for motorcycle handlebars designed to reduce structural vibrations and human exposure, Mechanical Systems and Signal Processing, 2012, Vol. 31, pp. 298–315

- [5] S. Agostoni, A. Barbera, E. Leo, M. Pezzola, M. Vanali, Investigation on motorvehicle structural vibrations caused by engine unbalances (1st part-Handlebar), Proceedings of the SEM Annual Conference June 1-4, 2009 Albuquerque New Mexico USA ©2009 Society for Experimental Mechanics Inc.
- [6] Jaimon Dennis Quadros, Suhas, Vaishak N.L, Shilpa.B, Study of vibration and its effects on health of a two wheeler rider, International Journal of Research in Engineering and Technology, Aug-2013, Volume: 02 Issue: 08.
- [7] Ali Israr, Hong Z. Tan, James Mynderse & George T.-C. Chiu, A psychophysical model of motorcycle handlebar vibrations, 2007 ASME International Mechanical Engineering Congress and Exposition.

BIOGRAPHIES



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