DESIGN, ANALYSIS AND FABRICATION OF SAVONIUS VERTICAL AXIS WIND TURBINE

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Abstract - In recent era, research and development activities in the field of renewable energy, especially wind and solar, have been considerably increased, due to the worldwide energy crisis and high global emission. The horizontal axis wind turbine cannot be used for household purpose. So, Savonius vertical axis wind turbine can be better option as it operate in low wind condition also. The choice for this model is to showcase its efficiency in varying wind conditions as compared to the traditional horizontal axis wind turbine and contribute to its steady growing popularity for the purpose of mass utilization in the near future as a reliable source of power generation.

Key Words: Renewable energy, VAWT, Savonius, Multi stage generator, Household, Handmade PM generator.

1. INTRODUCTION

The Savonius rotor is widely considered to be a drag-driven device. This indicates that the wind drag, acting on its blades, is the only driving force. However, it has been observed that at low angles of attack the lift force also contributes to the overall torque generation. Thus, it can be concluded that the Savonius rotor is not a solely drag-driven machine but a combination of a drag-driven and lift-driven device. Therefore, it can go beyond the limit of Cp established for the purely drag-driven machines (0.08). The idea of this turbine was proposed by Sigurd Johannes Savonius in 1922. However, for many years it was not widely applied. Recently, its popularity has steadily grown. This results from the increasing significance of urbanized areas, which have specific demands. The Savonius rotor seems to satisfy these particular expectations.

1.1 Advantages

1) Independence on wind direction, no additional control mechanisms are required
2) Ability to operate in a wide range of wind conditions (turbulence level, wind speed)
3) Electrical equipment can be placed at ground level
4) Low noise emission
5) High starting torque
6) Compact size
7) Simple and cheap construction.

All the above features make Savonius wind turbines suitable for the needs of residential use. However, Savonius turbines are not free from drawbacks such as relatively low efficiency and rotational velocity. Therefore, nowadays much effort is directed towards constantly seeking better designs that assure rotor performance improvement. Some of them also use CFD methods for this purpose.

1.2 Household Electricity Consumption

The electricity consumption was monitored in an house over the course of five months, and was then averaged out for a one month period. The calculations are shown below:

Average Monthly Electricity Consumption = 328.69 kWh/month
Average Yearly Energy Consumption = 328.69 x 12 = 3944.28 kWh/year or [3944.28/(24 x 365)] x 1000 = 450.26 W

For 10% of the electricity produced by the wind resource = 0.1 x 450.26 = 45.03 Watts. Therefore a 45.03 Watt machine is required. But, Considering 5% losses the final design is based on 50 watts. As we are using the multi stage generator we can predict that the power generation will be double i.e. 100 watts. So, with the design size of 50 watt turbine, the machine can give 100 watts output power.

2. DESIGN

2.1 Savonius Rotor Design

The key feature of this rotor is its simpler design. A Finish engineer Savonius introduced the Savonius rotor in 1920s. He has reformed the design of Flettener’s rotor by dividing a cylinder into half, along its central axis and relocating the two semi-cylindrical surfaces sideways. This shape is akin to “S” when viewed from top as shown in Fig. 1. These type of rotors may be of two, three or higher bladed systems and can be used in single- or multi-staged arrangements. The working principle is based on the difference of the drag force between the convex and the concave parts of the rotor blades when they rotate around a vertical shaft. This is chosen because of its simple construction and self-starting capacity at low wind speed also.
P = \( C_p \frac{V^3}{2} \) \( \rho \) (1)
Standard wind equation (derived from power coefficient analysis)
V = 10 m/s (assuming this is the wind speed);
\( \rho \) = 1.225 kg/m\(^3\) density of air at sea level and 15\(^o\)C;
Cp = 0.245 (24.5% from the standard Power Coefficient/Tip Speed Ratio diagram);
Power (P) = 50 Watts.
The equation below is used to calculate h:
\[ \text{Area} (A) = \text{height} (h) \times \text{diameter} (D) \] (2)

Table- 1: Height of rotor according to Diameter

<table>
<thead>
<tr>
<th>Diameter (m)</th>
<th>N(rpm)</th>
<th>A = 0.333 metre square, 50 Watts (5%) h (metres)</th>
</tr>
</thead>
<tbody>
<tr>
<td>D = 0.4</td>
<td>477.5</td>
<td>0.83</td>
</tr>
<tr>
<td>D = 0.5</td>
<td>382</td>
<td>0.67</td>
</tr>
<tr>
<td>D = 0.6</td>
<td>318.3</td>
<td>0.56</td>
</tr>
<tr>
<td>D = 0.7</td>
<td>272.2</td>
<td>0.48</td>
</tr>
<tr>
<td>D = 0.8</td>
<td>238.8</td>
<td>0.42</td>
</tr>
<tr>
<td>D = 0.9</td>
<td>212.2</td>
<td>0.37</td>
</tr>
<tr>
<td>D = 1.0</td>
<td>191</td>
<td>0.33</td>
</tr>
<tr>
<td>D = 1.1</td>
<td>173.6</td>
<td>0.303</td>
</tr>
</tbody>
</table>

The size of 0.83x0.4 (dxh) is chosen considering the height benefit and maximum number of rpm.

2.2 Generator Design
Instead of using readymade alternator we decided to use permanent magnet handmade generator according to design requirement. This PM generator is of multi stage. This is for increasing the output with same size rotor. The number of coils and magnet used in this are 12 and 24 in each stage respectively. The 3 phase connection is drawn from the coils and the this output can store in heavy duty battery. This type generator is generally designed when there is special requirement of power. Following figure shows the typical 3 phase connection of coils for PM magnet generator.

2.3 Component and Material
Rigid base – Mild steel
Inner fixed shaft- Mild steel (id=40mm, OD=50mm)
Outer rotary hollow shaft- Mild steel (id=80, OD=86)
Middle rotating disk- Mild steel
2 Ball Bearing- Bearing no. 6010(bore=50 mm)
Rotor blade- Aluminum alloy (No. of blade=4)
Magnets- Permanent earth magnet
Coils- Copper, Trapezoidal shape 50 turns
Battery- Lead acid heavy duty battery

2.4 CAD Model

Fig-1: 3 phase connection of 12 coils
Fig-2: Savonius Vertical axis wind turbine
3. ANALYSIS

3.1 Cfd

Computational fluid dynamics (CFD) is a useful design tool for wind power analysis. A large number of simulations can be performed, analyzed and optimized without investing in physical construction of many turbines with different geometrical configurations. Using CFD simulations, the torque and pressure on the rotor can be predicted. These can then be used to predict the turbine's power coefficient. Whenever the static analysis is to be done, the force exert on blade need to be calculate. So, Savonius rotor is the drag device but a little lift force also obtain in it. Hence, we have calculated the lift and drag force with the help of cfd analysis by using the blade cross section area as aero foil. Firstly defined the boundary condition then used inlet velocity input (10 m/s).

Graph-1: Pressure vs. position of blade

Force calculation,

\[ F = \sqrt{F_L^2 + F_D^2} \]

Lift force = \( F_L = \frac{1}{2} \rho V^2 A C_L \)

Drag force = \( F_D = \frac{1}{2} \rho V^2 A C_D \)

\( C_L = 0.385, C_D = 1.500 \)

Hence, \( F = 31.585 \text{ N} \)

The values of co-efficient of lift \( (C_L)\) and drag \( (C_D)\) are got from following graph:

Graph-2: Co-efficient of lift vs. No. of Iterations
3.2 Static structural
Static Structural analysis of blade is generally done to check whether it is sustainable or not in the defined working environment. On the basis of calculated force, the analysis of blade is done by using aluminum alloy as the blade material. The following figure shows the analysis report:

Material = Aluminum alloy

**Graph-3:** Co-efficient of drag vs No. of Iterations

**Fig-5:** Deformation in Savonius turbine blades

**Fig-6:** Equivalent Stress in Savonius turbine blades

**Fig-7:** Shear stress on XY plane in Savonius blades
3.3 Dynamic
Dynamic analysis is done to check the designed model under variable condition of load. In this case varying condition are refers to the change in velocity of air with respect to time.

![Dynamic Analysis](image)

**Fig-8:** Total Deformation in blades

**Fig-9:** Equivalent stress in blades

**Fig-10:** Shear stress on XY plane in blades

4. SOFTWARE TOOL
Today's world is deal with technology in every field. There are certain software in market which is used for design and analysis of the product to determine its behavior under defined conditions. It is better to use these software before directly go for manufacturing. It helps to gives errorless design and save the time and money also.

4.1 Design
Design of model is done in creo software. This is more accurate for 3D modeling.

4.2 Analysis
It is done in Ansys software. Various types of analysis such as cfd,static,dynamic can be done in this software.

5. METHODOLOGY

<table>
<thead>
<tr>
<th>Table-2: Applied Procedure for Savonius VAWT</th>
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</thead>
<tbody>
<tr>
<td>[1] Average Monthly consumption of electricity in an house</td>
</tr>
<tr>
<td>[2] Percentage of electricity to be contribute to the house by VAWT</td>
</tr>
<tr>
<td>[3] Size of the turbine required for that power generation</td>
</tr>
<tr>
<td>[5] Design of the multi stage hand made permanent magnet generator</td>
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6. RESULTS AND DISCUSSION
The vertical axis wind turbine is a small power generating unit with the help of free source of wind energy.It is designed under consideration of household use.The analysis results of the wind turbine governs that the design is under safe limit.It is check in analysis for worst condition working by using velocity 10 m/s.

6.1 Analysis Results
Blade Material= Aluminum alloy
1. Computational fluid dynamics (CFD)
   $C_L = 0.3850 \times C_D = 1.5000$
   $F_L = 7.852 \text{ N}, F_D = 30.594 \text{ N}$
   $F = 31.585 \text{ N}$

2. Static Structural
   Total Deformation - Max = 2.1069 mm
   Min = 0 mm
   Equivalent stress - Max = 4.915 Mpa
   Min = 0 Mpa
   Shear stress - Max = 0.165 Mpa
   Min = -0.165 Mpa
   UTS = 310 Mpa

3. Dynamic
   Total Deformation - Max = 2.002 mm
   Min = 0 mm
   Equivalent stress - Max = 4.671 Mpa
   Min = 0 Mpa
   Shear stress - Max = 0.1568 Mpa
   Min = -0.1568 Mpa

From the above obtained value of the analysis report, it can be conclude that the design is safe and within the permissible limit. The field testing of the turbine will be carried out to get the power output result.

7. CONCLUSION
There are number of sources for generation of power but in the recent years wind energy shown its potential as the clean source of energy and contributing to the high energy demands of the world. Vertical axis wind turbine is the best option for the area which are under load sheading.

The output from the turbine is used to charge a heavy duty battery. This can be beneficial than inverter back up. Following are the some conclusions drawn from this project:

1. At least 10% power of the consumption can be fulfilled by this set up.
2. Multi stage generator is the double generation concept with the same size rotor.
3. Gear arrangement can increase the number of rpm in case of low wind speed.
4. This turbine is generally suitable for 8 to 10m of height above ground level. Because at ground level velocity of air is very less.
5. Combination of alternator with gear arrangement can be used to increase output but unnecessarily it will increase the cost of machine.
6. Considering the all-weather point of view the material should be non-corrosive.
7. The alternate option for turbine blade material is reinforced glass fibre because of its more elastic nature but it is costlier than aluminium alloy.
8. The cost of the machine should be as minimum as possible. So that it will be economical for everyone to purchase.

REFERENCES


